

SENSIBLE EQUIPMENT COMPANY

# TRICOIL®

*The Sensible Solution*

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PSYCHROMETRIC CHART  
NORMAL TEMPERATURE

I-P Units  
SEA LEVEL

BAROMETRIC PRESSURE : 29.921 INCHES OF MERCURY

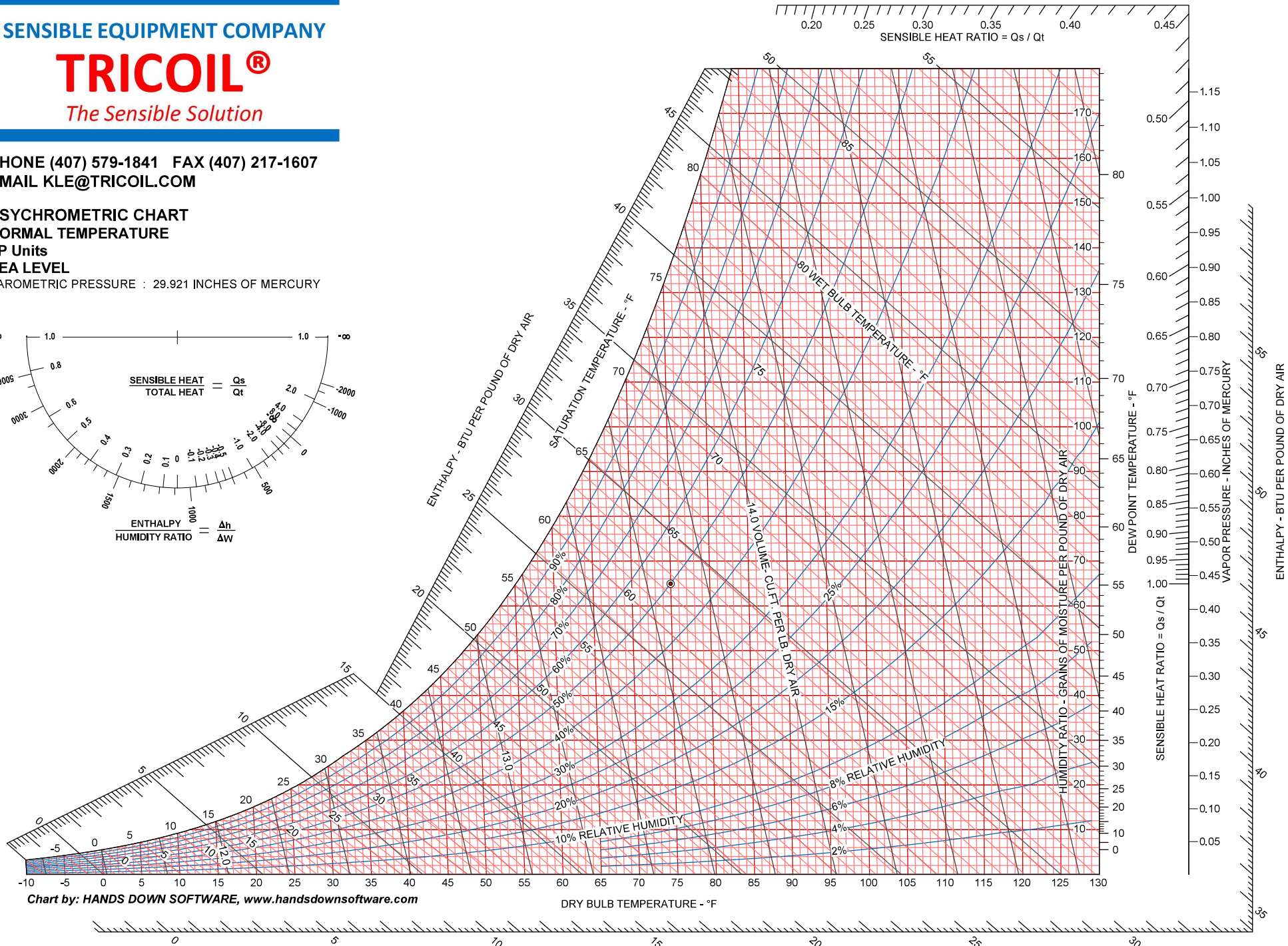
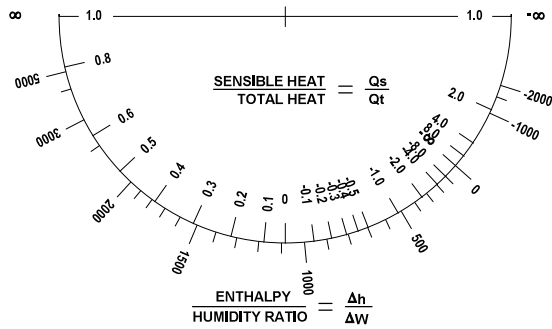


Chart by: HANDS DOWN SOFTWARE, [www.handsdownsoftware.com](http://www.handsdownsoftware.com)

## PROCEDURE FOR USING THE PSYCHROMETRIC CHART TO MAKE TRICOIL<sup>®</sup> SELECTIONS

The application of a typical TRICOIL system is illustrated in the following example. The TRICOIL<sup>®</sup> Method is a proprietary system and is protected by United States of America patent laws, patent numbers 5,181,552; 5,228,302; 5,337,577; and 5,493,871. Other USA and foreign patents are pending. Contact Sensible Equipment Company, P.O. Box 669, Goldenrod, Florida 32733. Phone Number (407) 296-8068 Fax Number (407) 217-1607.

### EXAMPLE:

- All projects start with a set of criteria which are used as a basis for calculating the room design cooling load. For this example the following criteria was used.

Design Criteria:	Room Conditions	76°Fdb/45%rh
	Outside Air Conditions	95°Fdb/80°Fwb
	Supply Air Relative Humidity	70% rh
	Outside Air Volume	2000 cfm

- Calculate the room sensible heat gain ( $Q_{RSH}$ ) and the room latent heat gain ( $Q_{RLH}$ ). Also estimate the supply air ( $\Delta T_d$ ) and return air ( $\Delta T_e$ ) temperature rise due to fan, motor and duct heat gain. The following information was calculated using ASHRAE procedures:

$$\begin{aligned} Q_{RSH} &= 176,000 \text{ btu/hr} & \Delta T_d &= 2^\circ\text{F} \\ Q_{RLH} &= 44,000 \text{ btu/hr} & \Delta T_e &= 1^\circ\text{F} \end{aligned}$$

- Calculate the room sensible heat factor (RSHF) as follows:

$$\begin{aligned} \text{Room Sensible Heat Factor, RSHF} &= Q_{RSH} / (Q_{RSH} + Q_{RLH}) \\ &= 176000 / (176000 + 44000) = 0.80 \end{aligned}$$

- Draw the room sensible heat factor line on the protractor and transfer the line to the chart starting at the room conditions and terminating at the 70% relative humidity line. The point of intersection is the room supply air condition.

- Calculate supply air volume as follows:

$$\begin{aligned} \text{Supply Air Volume, CFM}_{SA} &= Q_{RSH} / [1.1 \times (T_1 - T_2)] \\ &= 176000 / 1.1 \times (76-60) \\ &= 10,000 \text{ cfm} \end{aligned}$$

- Calculate reheat required ( $\Delta T_c$ ) from TRICOIL Loop:

$$\Delta T_c = T_2 - T_4 - \Delta T_d = 60 - 50 - 2 = 8^\circ\text{F}$$

- Determine mixed return air / outside air condition: Neglecting the effects of air density, the solution can be determined as follows:

$$\begin{aligned} T_7 &= (T_1 + \Delta T_{RA}) \times (\text{CFM}_{RA} / \text{CFM}_{SA}) + T_6 \times (\text{CFM}_{OA} / \text{CFM}_{SA}) \\ &= 77 \times (8000/10000) + 95 \times (2000/10000) = 80.6^\circ\text{F db} \end{aligned}$$

- Determine precooled air condition:

$$\begin{aligned} \text{Precooling } (\Delta T_a) &= \text{Reheat } (\Delta T_c) = 8^\circ\text{F } \Delta T \\ T_8 &= T_7 - \Delta T_a = 80.6 - 8 = 72.6 \end{aligned}$$

- Determine enthalpy (h) for calculating primary cooling:  
Project state points  $T_4$  and  $T_8$  through enthalpy lines.  
 $\Delta H_b$  (differential enthalpy) =  $h_8 - h_4$   
 $\Delta H_b = 29.1 - 20.2 = 8.9 \text{ btu/lb dry air}$

- Calculate coil requirements.

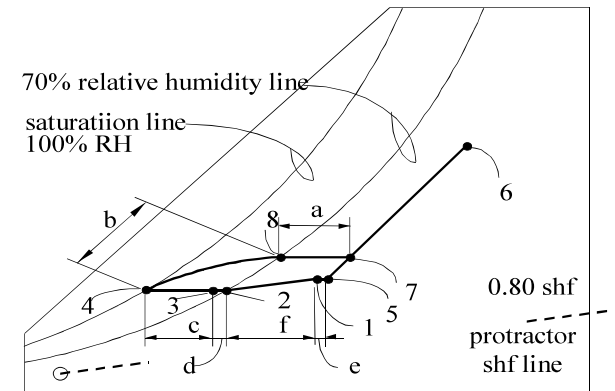
$$\begin{aligned} \text{Precooling: } Q_{PC} &= 1.1 \times \text{CFM}_{SA} \times \Delta T_a \\ &= 1.1 \times 10000 \text{ cfm} \times 8^\circ\text{F } \Delta T \\ &= 88,000 \text{ btu/hr} \end{aligned}$$

$$\begin{aligned} \text{Primary Cooling: } Q_{PR\text{Sens}} &= 1.1 \times \text{CFM}_{SA} \times \Delta T_b \\ &= 1.1 \times 10000 \times (72.6 - 50) \\ &= 248,600 \text{ btu/hr} \end{aligned}$$

$$\begin{aligned} Q_{PR\text{total}} &= 4.45 \times \text{CFM}_{SA} \times \Delta H_b \\ &= 4.45 \times 10000 \text{ cfm} \times (29.1 - 20.2) \\ &= 396,000 \text{ btu/hr} = 32.6 \text{ tons} \end{aligned}$$

$$\begin{aligned} \text{Reheat: } Q_{RH} &= 1.1 \times \text{CFM}_{SA} \times \Delta T_c \\ &= 1.1 \times 10000 \text{ cfm} \times 8^\circ\text{F } \Delta T \\ &= 88,000 \text{ btu/hr} \end{aligned}$$

**PSYCHROMETRIC CHART** The following chart indicates the process and state points of the TRICOIL<sup>®</sup> operation in the above example.



### STATE POINT

1. Room Air:	76°Fdb/45%rh
2. Supply Air:	60°Fdb/54.2°Fwb
3. Leaving Reheat Coil:	58°Fdb/53.5°Fwb
4. Leaving Primary Coil:	50°Fdb/49.8°Fwb
5. Return Air:	77°Fdb/62.4°Fwb
6. Outside Air:	95°Fdb/80.0°Fwb
7. Mixed Air (R.A. and O.A.):	80.6°Fdb/66.4°Fwb
8. Air Leaving Precooling Coil:	72.6°Fdb/63.8°Fwb

### PROCESS

a. Precooling:	$Q_{PC} = 88,000 \text{ btu/hr}$
b. Primary Cooling:	$Q_{PR} = 396,000 \text{ btu/hr}$
c. Reheat:	$Q_{RH} = 88,000 \text{ btu/hr}$
d. Supply Air Heat Gain: (Duct, Fan, and Motor)	$\Delta T_{SA} = 2^\circ\text{F}$
e. Return Air Heat Gain: Duct	$\Delta T_{RA} = 1^\circ\text{F}$
f. Room Heat Gain: Sensible:	$Q_{RSH} = 176,000 \text{ btu/hr}$
Latent:	$Q_{RLH} = 44,000 \text{ btu/hr}$